



Modeling of Composites in LS-DYNA

- **Some Characteristics of Composites**
- **Orthotropic Material Coordinate System**
- **User-defined Integration Rule for Shells**
- **Output for Composites**
- **Some Characteristics of Several Composite Material Models in LS-DYNA**
- **Closing Recommendations**



Two Types of Composites

- **Advanced composites have stiff, high strength fibers bound in a matrix material.**
 - Each layer/lamina/ply is orthotropic by nature as the fibers run in a single direction.
 - Usually, an advanced composite section will have multiple layers and each lamina within the stack will have the fibers running in a different direction than in the adjacent lamina.

- **A sandwich composite section has laminae which may be individually isotropic but the material properties and thickness may vary from lamina to lamina.**
 - A foam core composite is a particular type of sandwich composite where a thick, soft layer of foam is sandwiched between two thin, stiff plies.



Orthotropic Materials in LS-DYNA

- Orthotropic material constants are defined in the *material coordinate system*.
- The *material coordinate system* must be initially established for each orthotropic element and, in the case of shells, for each through-thickness integration point as well. This orientation comes from three sources.
 - In the material definition (*mat)
 - See description of “AOPT” in User’s Manual under *mat_2 (orthotropic_elastic)
 - In the section definition (*section_shell)
 - A “beta” angle is given for each integration point
 - Optionally, in the element definition (*element_shell_beta, *element_solid_ortho)



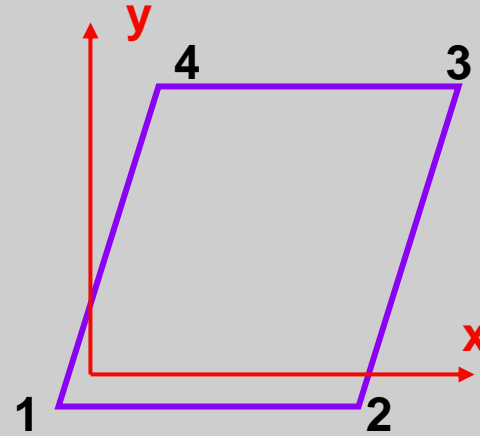
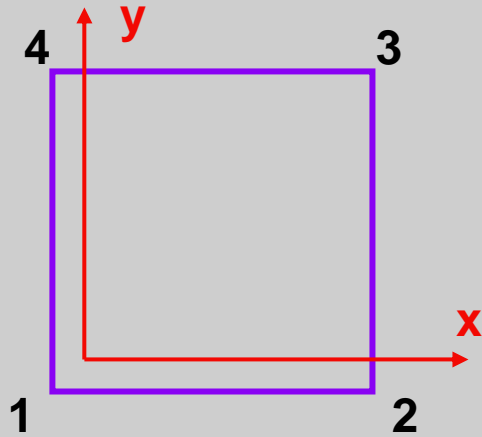
Orthotropic Materials in LS-DYNA

- As the solution progresses and the elements rotate and deform, the *material coordinate system* is automatically updated, following the rotation of the *element coordinate system*.
 - The orientation of the *material coordinate system* and thus response of orthotropic shells can be very sensitive to in-plane shearing deformation and hourglass deformation, depending on how the *element coordinate system* is established.
 - To minimize this sensitivity, “Invariant Node Numbering”, invoked by setting $INN = 2$ (shells) or 3 (solids) in ***control_accuracy** is highly recommended.



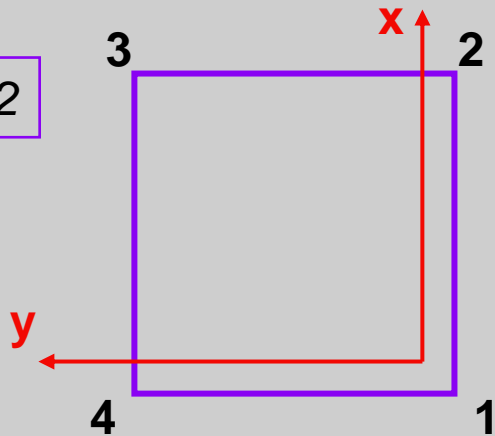
Without Invariant Node Numbering (N1-to-N2 establishes element x-direction)

Case 1

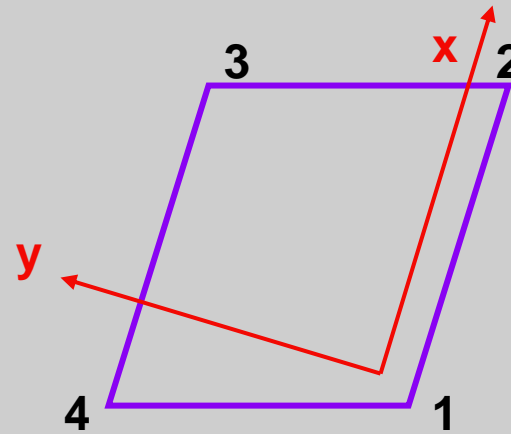


Element rotation = 0

Case 2



Undeformed



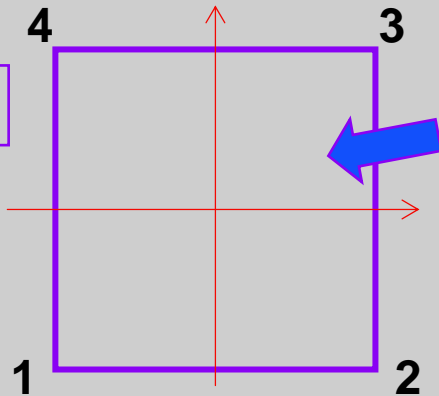
Deformed

Element rotation = - 20°

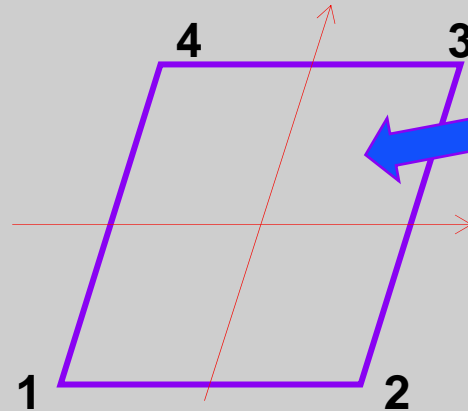


With Invariant Node Numbering (based on element bisectors)

Case 1



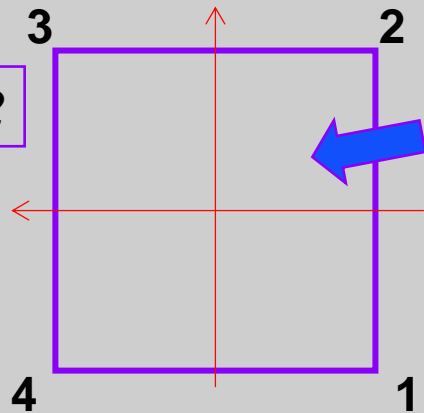
Local x
 $= (0+90)/2 - 45$
 $= 0^\circ$



Local x
 $= (0+70)/2 - 45$
 $= -10^\circ$

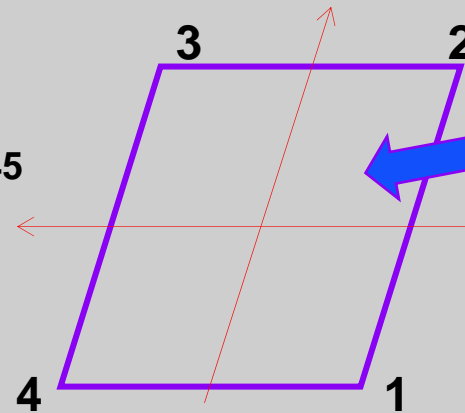
Element rotation
 $= -10 - 0 = -10^\circ$

Case 2



Local x
 $= (90+180)/2 - 45$
 $= 90^\circ$

Undeformed



Local x
 $= (70+180)/2 - 45$
 $= 80^\circ$

Element rotation
 $= 80 - 90 = -10^\circ$

Deformed



User-Defined (Through-Thickness) Integration

- Gaussian or Lobatto integration rules have pre-established integration point locations and weights (NIP ≤ 10).
 - **Lobatto includes integration points on the outside surfaces**
- Trapezoidal integration has equally spaced integration points.
- For composites, the user may need to define his/her own integration point locations and weights (corresponding to ply thicknesses) and may need to reference a different set of material constants for each integration point.



User-Defined Integration (970)

```

*PART
material 1
    1      1      11

*PART
material 2
    2      1      12
$-----1-----2-----3-----4-----5-----6-----
*SECTION_SHELL
    1      2
    18.000000 18.000000 18.000000 18.000000
*mat_layered_linear_plasticity
11, 2.7e-6, 73.4, 0.32, 1e9

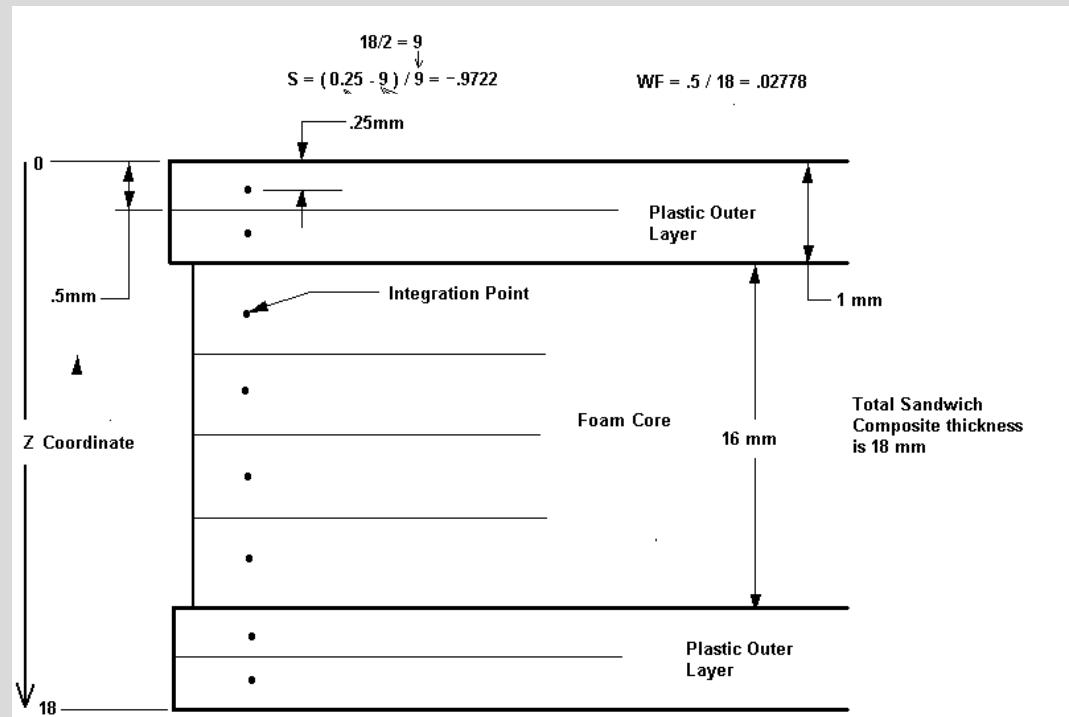
*mat_layered_linear_plasticity
12, 6.3e-7, 0.286, 0.3, 1e9

*INTEGRATION_SHELL
20,8,0
-.9722, .02778, 1
-.9167, .02778, 1
-.6667, .22222, 2
-.2222, .22222, 2
.2222, .22222, 2
.6667, .22222, 2
.9167, .02778, 1
.9722, .02778, 1

*ELEMENT_SHELL
    1      1      1      2      33
    2      1      2      3      34
    
```

Negative value indicates user integration rule

-20



Sandwich Composite Material Cross-Section Example



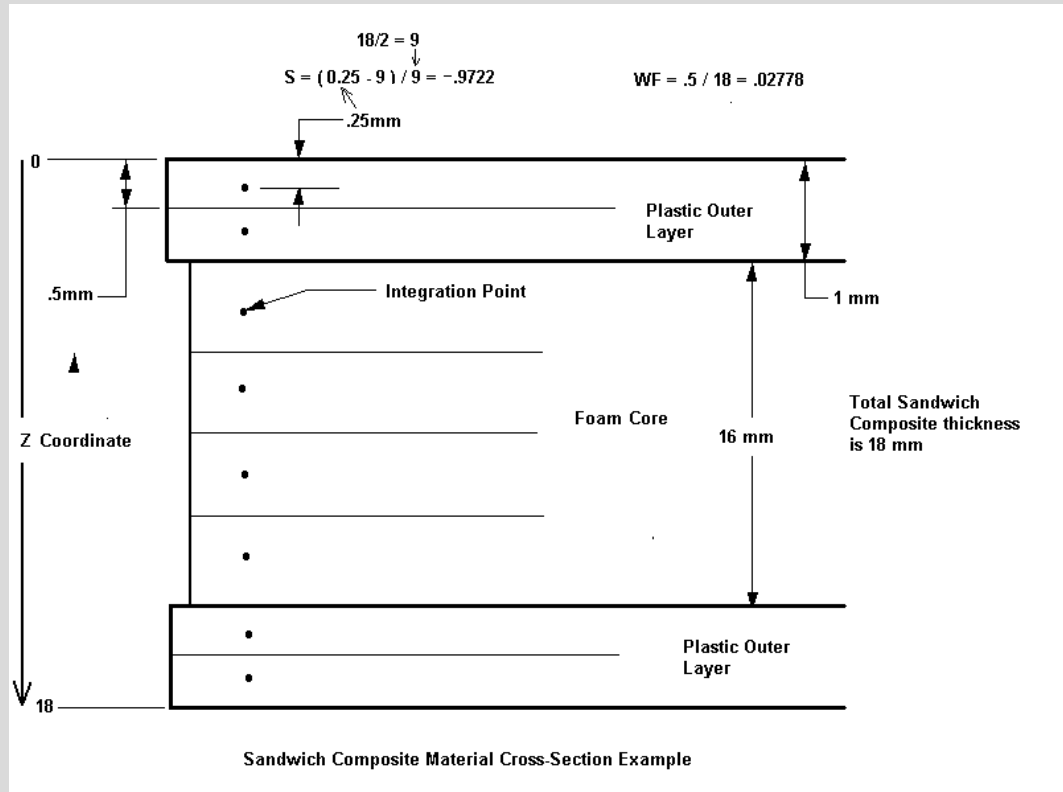
User-Defined Integration (971)

```
$ no *section command needed
$ thickness is sum of thick values given in *PART_COMPOSITE
$ no need for multiple *PART commands
$
```

```
*PART_COMPOSITE
$ pid, elform
1, 2
$ mid, thick, beta,,mid,thick,beta
11, 0.5,,, 11, 0.5
12, 4.0,,, 12, 4.0
12, 4.0,,, 12, 4.0
11, 0.5,,, 11, 0.5
*mat_layered_linear_plasticity
11, 2.7e-6, 73.4, 0.32, 1e9
```

```
$ NOTE: foam core could use a different
$ material model (971)
*mat_layered_linear_plasticity
12, 6.3e-7, 0.286, 0.3, 1e9
```

```
*ELEMENT_SHELL
1 1 1 2 33
2 1 2 3 34
```





Output for Composites

- For composite material models, stresses (and strains) will be written in the material coordinate system rather than the global coordinate system *if* CMPFLG (and STRFLG) is set to 1 in *database_extent_binary.
 - **Useful option for postprocessing of fiber and matrix stresses.**
- Set MAXINT in *database_extent_binary to the total number of through-thickness integration points in your composite shell. By default, stresses only at the top, bottom, and middle integration points are written.



Output for Composites

- Some composite material models have “extra history variables” that help to track modes of failure in each integration point. (See material documentation in the User’s Manual for details.)
 - **NEIPS (shells) or NEIPH (solids) in *database_extent_binary should be set to the number of extra history variables needed. For example, if you want to track the damage parameter in mat_054, set NEIPS to 6.**



Composite Material Models

- ***mat_2 (elastic_orthotropic)**
 - 9 elastic constants (solids); 6 elastic constants (shells).
 - Total Lagrangian formulation (okay for large elastic deformations).
 - No failure criteria.

- Each of the following orthotropic materials offer a particular brand of fiber/matrix damage and failure criteria. Up to 5 strength values are given (XT, XC, YT, YC, SC).
 - ***mat_22 (composite_damage)**
 - ***mat_54,55 (enhanced_composite_damage)**
 - ***mat_58 (laminated_composite_fabric)**
 - *mat_158 includes strain rate effects
 - ***mat_59 (composite_failure(_shell, _solid)_model)**
 - Can be used with shells or solids



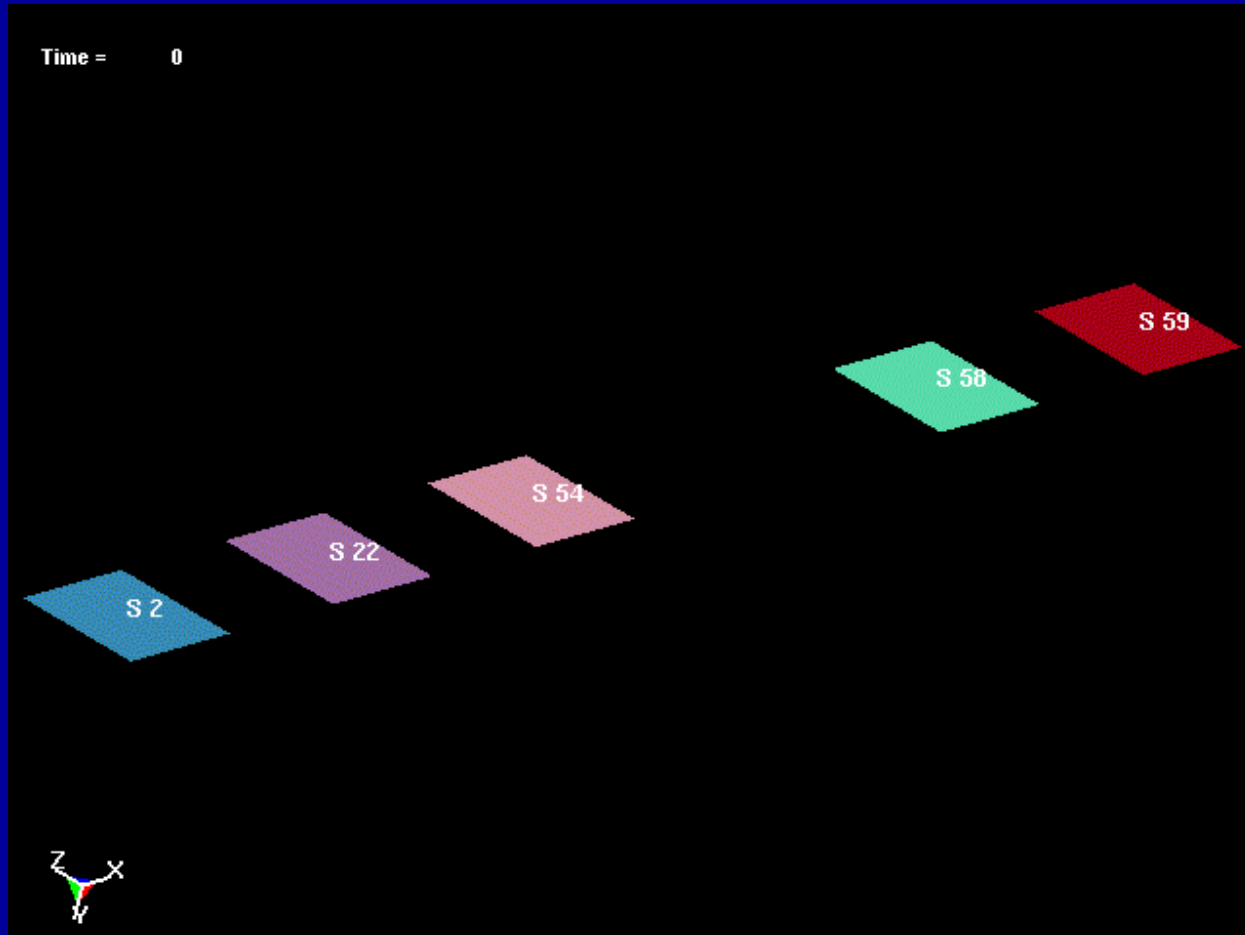
Composite Material Models

- The paper "Crashworthiness Analysis with Enhanced Composite Material Models in LS-DYNA - Merits and Limits", Schweizerhof et al, 5th International LS-DYNA User's Conference (1998) provides some insight into several composite material models in LS-DYNA, including mat_54, mat_58, and mat_59. This paper is available as a PDF file.



Comparison of Several Composite Material Models

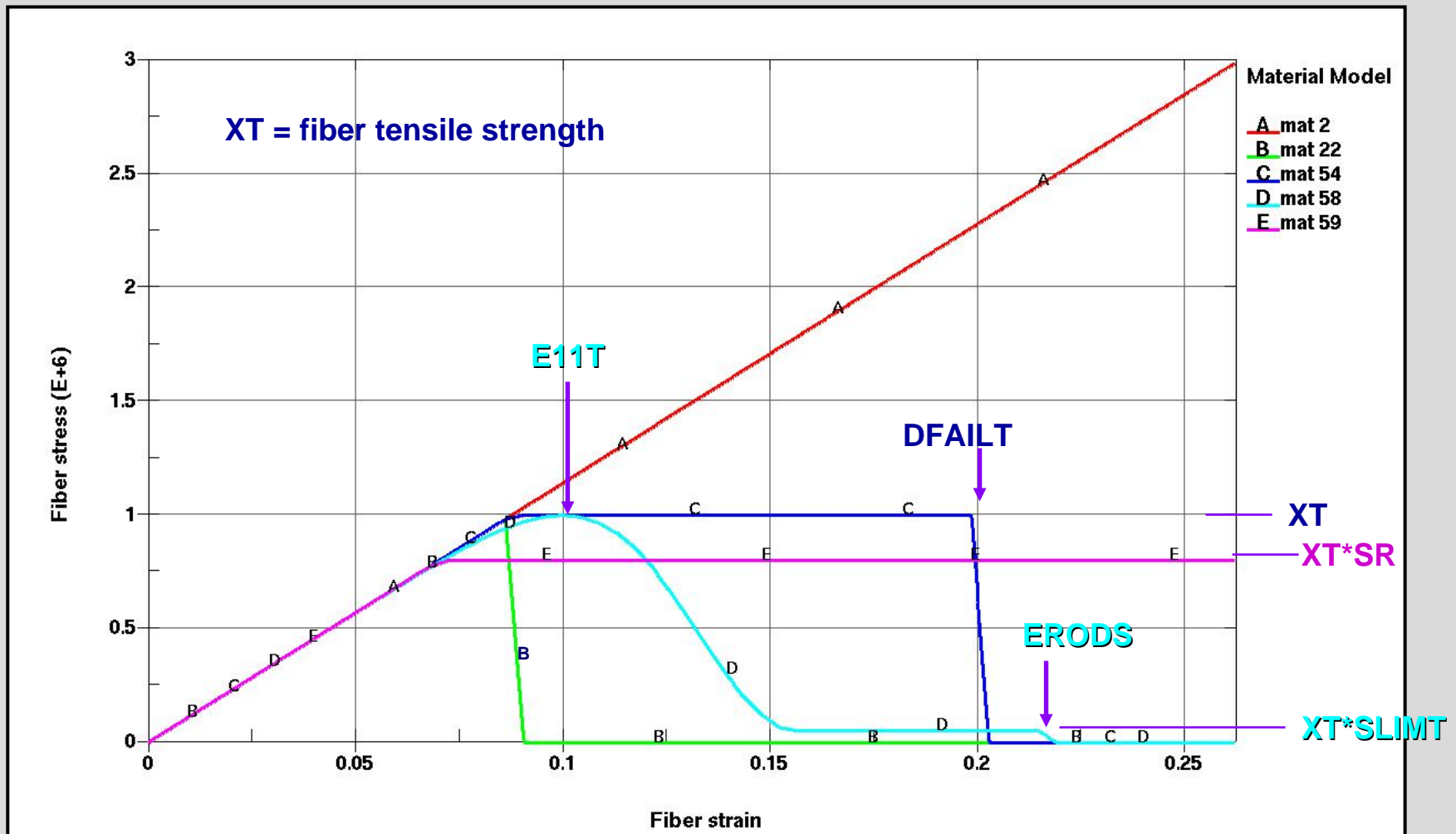
- Uniaxial Tension in Fiber Direction





Comparison of Several Composite Material Models

■ Uniaxial Tension in Fiber Direction





Laminated Shell Theory

- Use of Laminated Shell Theory (LST) is important if a composite shell has layers of dissimilar materials.
 - LST corrects for the incorrect assumption of uniform constant shear strain through the thickness of the shell.
 - Without LST, a sandwich composite will generally be much too stiff.
 - LAMSHT=1 in *control_shell invokes LST for material models 22, 54, 55, 76
 - *Mat_layered_linear_plasticity (114) is a plasticity model much like mat_024 but which includes LST.



Composite Material Models

- ***mat_116 (composite_layup)**
 - Orthotropic elastic *resultant* formulation (no stresses calculated)
 - Very efficient for large number of layers
 - Requires *integration_shell
 - Material constants can vary from layer to layer
 - Does NOT use laminated shell theory (not good for foam core/sandwich composites)



Composite Material Models

- ***mat_117 (composite_matrix)**
- ***mat_118 (composite_direct)**
 - **Resultant formulation (no stresses calculated)**
 - **21 coefficients of symmetric stiffness matrix are input directly**
 - Stiffness coefficients in 117 given in material coord system
 - Stiffness coefficients in 118 given in element coord system (less storage req'd)
 - **Shell thickness is inherent in stiffness matrix. Thus uniform thickness of part is mandatory.**



Composite Material Models

- ***mat_161 (composite_msc)**
 - Proprietary model from Materials Sciences (requires license add-on)
 - Available for solids only
 - Offers fiber shear and fiber crush failure criteria
 - Can predict delamination
 - *mat_162 like *mat_161 but adopts damage mechanics approach for softening after damage initiation



A Word about Delamination

- Shells do not have σ_{zz} component of stress and thus are not well-suited to rigorous study of composite delamination.
- Delamination may be approximated using multiple layers of shells tied with `*CONTACT_..._TIEBREAK` in which failure of contact represents delamination.



Closing Recommendations

- Most composites do not stretch significantly before breaking. To promote numerical stability, shell thinning option should NOT be invoked. Leave ISTUPD in *control_shell set to zero.
- 'Noise' in response can be mitigated by stiffness damping in some cases. See *damping_part_stiffness.
- Shell bulk viscosity (*hourglass, ITYPE=-1) may aid stability in compressive modes of response.